Total Number of Pages : 03				
210	B	TION & AIR CON RANCH : MECH Time : 3 Hours Max Marks : 100 Q.CODE : C227	DITIONING	210
210	The figures in the r	• •	-	210
210	Part – A (Answer all the que	estions)	210
Q1	Answer the following question		•	(2 x 10)
a)	Refrigerant used should be such (i) Greater than the temperatur (ii) Less than the temperature r (iii) Equal to the temperature ref	e required equired	ling point is	
210	(iv) None 210 210	210	210	210
b)	The wet bulb temperature of air(i) less than dew point tempera(iii) greater than dry bulb tempe	ature (dpt) (ii)	equal to dpt	dpt
c)	Ammonia as a refrigerant has _ (i) high (iii) medium	volume (ii) low (iv) all of		
²¹ d)	In ice plant ^{_210} refriĝera (i) R11 (iii) NH ₃	ant is commońlŷ us (ii) R21 (iv) R134		210
e)		()	mpressors I	
f) 210	A refrigerator is having COP of 210 If the device is used as heat pu	4. T _{max} /T _{min}	210	210
g)	1 kg/s of air at 15°C is flowing 40°C for winter air-conditioning BPF and Efficiency of the coil ar	through a heating system. The outlet	coil having temperature of air is	
h)	A mixture of dry air and water pressure of 736mm Hg. The dev Pv, RH are ar	w point temperature	e is 15C. P	
²¹⁰ i)	In VARS, generator, condenser and 20°C respectively. Obtain th	and evaporator are	e maintained at 80°C	C, 40°C ²¹⁰
j)	During the compression proce refrigerant gas characteristics of (i) increase, some (iii) increase, all	fgas (ii) decr	es.	of the

Q2		Answer the following questions : Short answer type :	(2 x 10)
	a)	Define COP and TR.	
	210)	Why is Reversed Carnot cycle not used in actual practice? 210 210	
	C)	Draw the P-h and T-s chart for vapor compression system (wet compression).	
	d)	How are the refrigerants numbered?	
	e)	Mention the types of expansion devices used in refrigeration systems.	
	f)	What are secondary refrigerants? Give one example of a secondary refrigerant.	
	²¹⁰ g)	Show cooling & humidification and heating ²¹ & dehumidification process on ²¹⁰ psychometric chart.	
	h)	Explain various heat loads to be considered for cooling load calculations.	
	i)	Differentiate between absorbents and adsorbents.	
	j)	Define GSHF and RSHF.	
	210	Part – B (Answer any four questions) 210 210	
Q3	a)	Derive an expression for COP of a Bell-Colemann cycle (Reversed Bryton). Discuss the merits and demerits of open and close Bell-Colemann cycle	(10)
	b)	Explain the working of air craft cooling system (any one type) with schematic and T-s diagram	(5)
Q4	a) 210	A Freon 12 vapor compression system operating at a condenser temperature of 40°C and an evaporator temperature of 0°C develops 15°TR. Using p-h ²¹⁰ chart, determine :	(10)
		 (i) The discharge temperature and mass flow rate of the refrigerant circulated (ii) The theoretical piston displacement of the compressor and piston displacement per TR of refrigeration 	
		(iii) The theoretical horsepower of the compressor and horsepower per TR	
		(iv) The heat rejected in the condenser	
	210	(v) The Carnot COP and actual COP of the cycle. 210 210	
	b)	Discuss the effect of the following on the performance of vapor compression system	(5)
		(i) Effect of suction pressure(ii) Effect of delivery pressure(iii) Effect of superheating(iv) Effect of subcooling	
Q5	a)	A two stage vapor compression refrigeration system with a flash chamber	(10)
	210	operates with amoonia as refrigerant. The evaporator and condenser temperatures are -30 and 40°C respectively. If the capacity of the plant is 30 TR, estimate the total work of compression and the COP.	. ,
		Had the compression been done in a single stage, what would have been the percentage increase in the work of compression? What is the percentage of increase in COP owing to the staging of the compression process?	
	b)	Describe multistage compression system with inter-cooling with neat sketch and T-s, h-s diagrams	(5)
	210	210 210 210 210 210	

Q6	a)	With neat sketches explain the Aqua-Ammonia vapor absorption system. Mention differences between vapor compression and vapor absorption system			(8)	
	²¹⁰ b)	•	210	(7)	21	
Q7	a)	pressure of 736 mm Hg. The dew point temperature is 15°C. Find : (i) Partial pressure of water vapor				
	210	 (ii) Relative humidity (iii) Degree of saturation ²¹⁰ ²¹⁰ ²¹⁰ ²¹⁰ ² (iv) Specific humidity (v) Specific enthalpy of moist air (vi) Specific volume of air per kg of dry air 	210		21	
	b)	Obtain the relationship between relative humidity and degree of saturation as below		(5)		
	210	$\phi = \frac{\frac{210}{\mu} \frac{\mu}{210}}{1 - (1 - \mu) \frac{p_s}{p_b}}$ 210 210 2	210		2	
			((10)		
Q8	a) 210	 A building has the following calculated cooling loads; RSH gain:310 kW, RLH gain²¹100 kW ²¹⁰ ²¹⁰ ²¹⁰ ²¹⁰ ²¹⁰ The space to be maintained at 25°C DBT and 50% RH. Out door air is 28°C and 50% RH and 10% by mass of air supplied to the building is outdoor air. If the air supplied to the space is not to be at a temperature lower than 18°C, find : (i) Minimum amount of air supplied in m³/s. (ii) Volume flow rates of return air, exhaust air and outdoor air 			2	
	210	(iii) State and volume flow rate of air entering the cooling coil at (iv) Capacity, ADP, BPF and SHF of the cooling coil.	210		2	
	b)	Explain a neat diagram, the winter air conditioning system		(5)		
Q9	a) b)	Explain Actual Vapor Compression System with p-h and T-s chart		(5) (5) (5)		
	_ C)					

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